

CFD Analysis of a Lean Premixed Gas Turbine Combustor for H2 Applications

Franco Agustín Rizzi
MSc Mechanical Engineering

Daniela Anna Misul, Simone Salvadori, Salvatore Carusotto, DENERG, Politecnico di Torino, Italy



**Politecnico
di Torino**

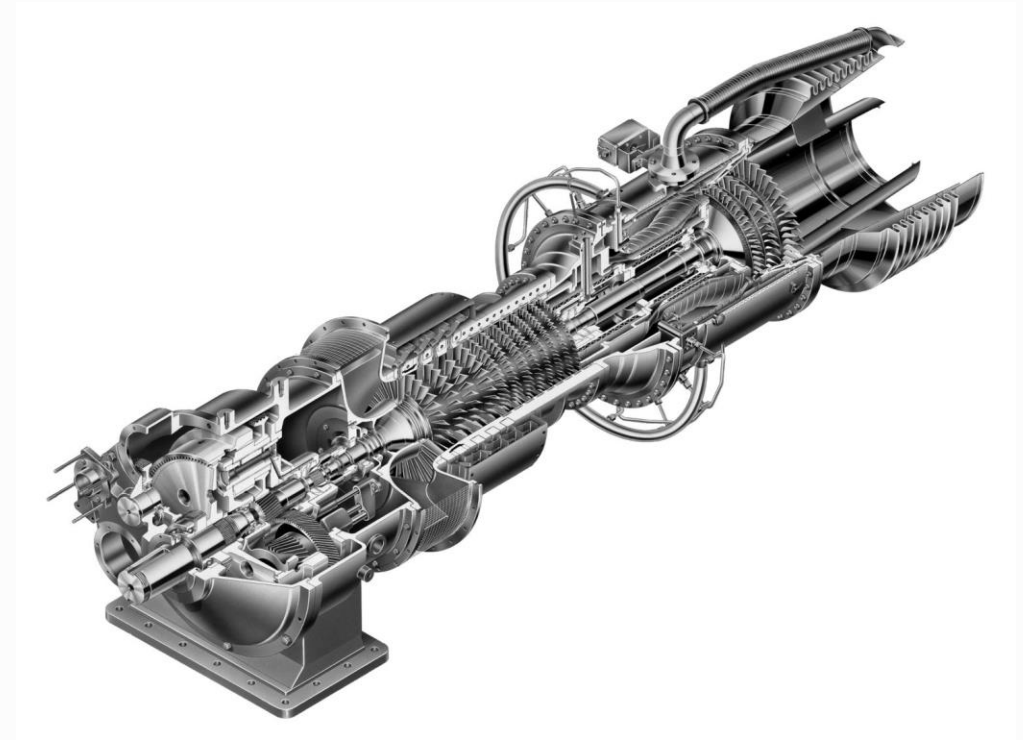
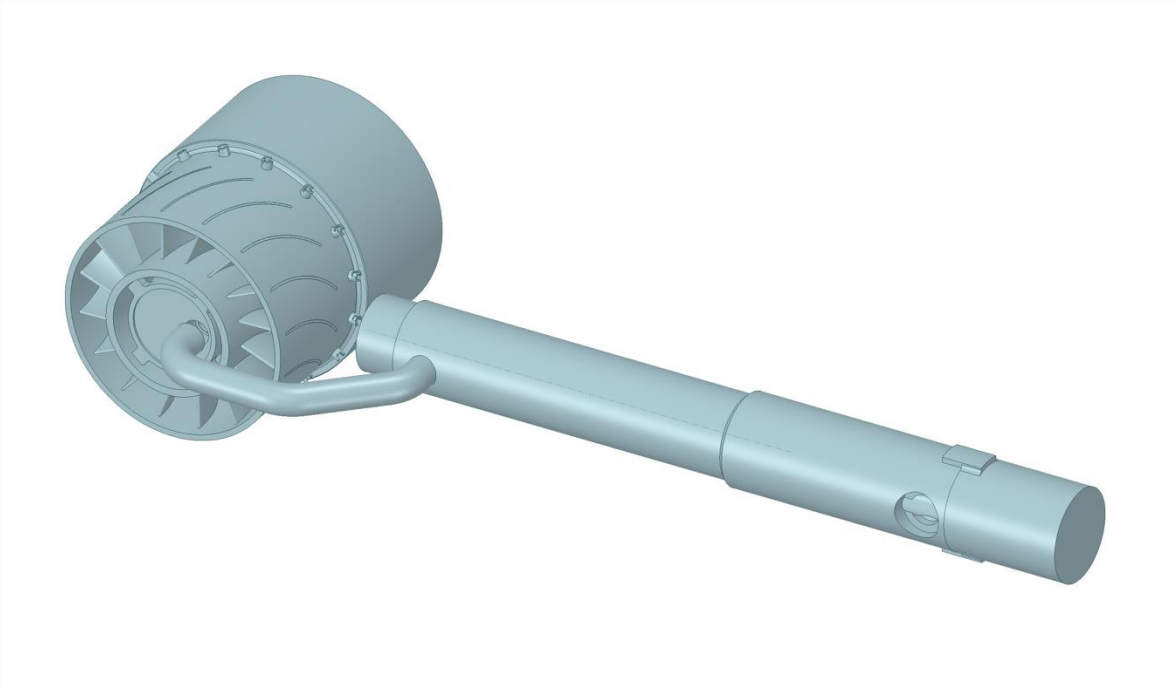
Department of Energy
"G.Ferraris"



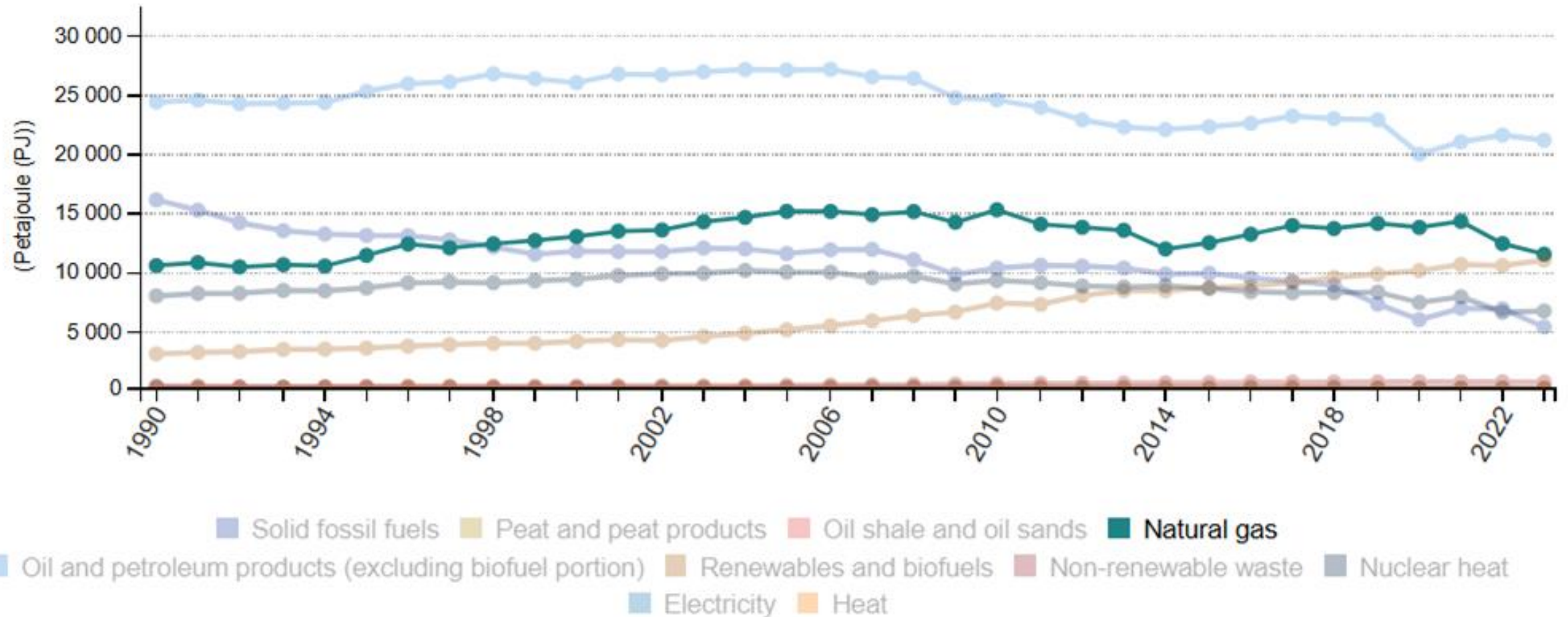
EthosEnergy

Motivation

EthosEnergy S.P.A developed an injector that allows the introduction of hydrogen – methane mixture to the combustion chamber of a gas-turbine

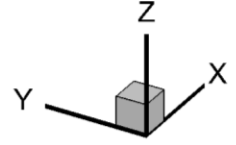
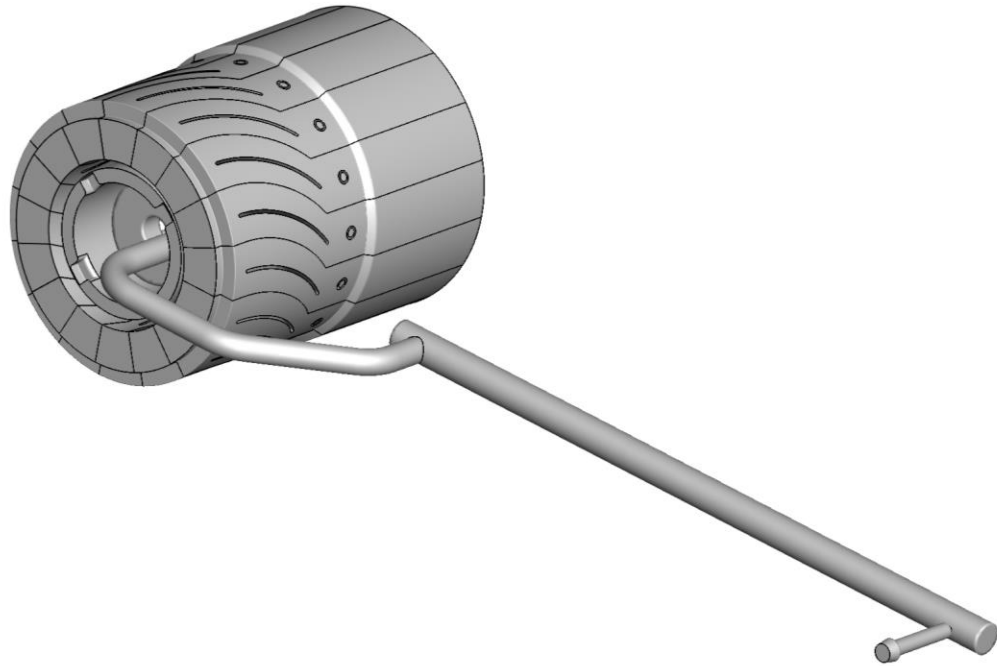


Gross available energy by fuel, EU, 1990-2023

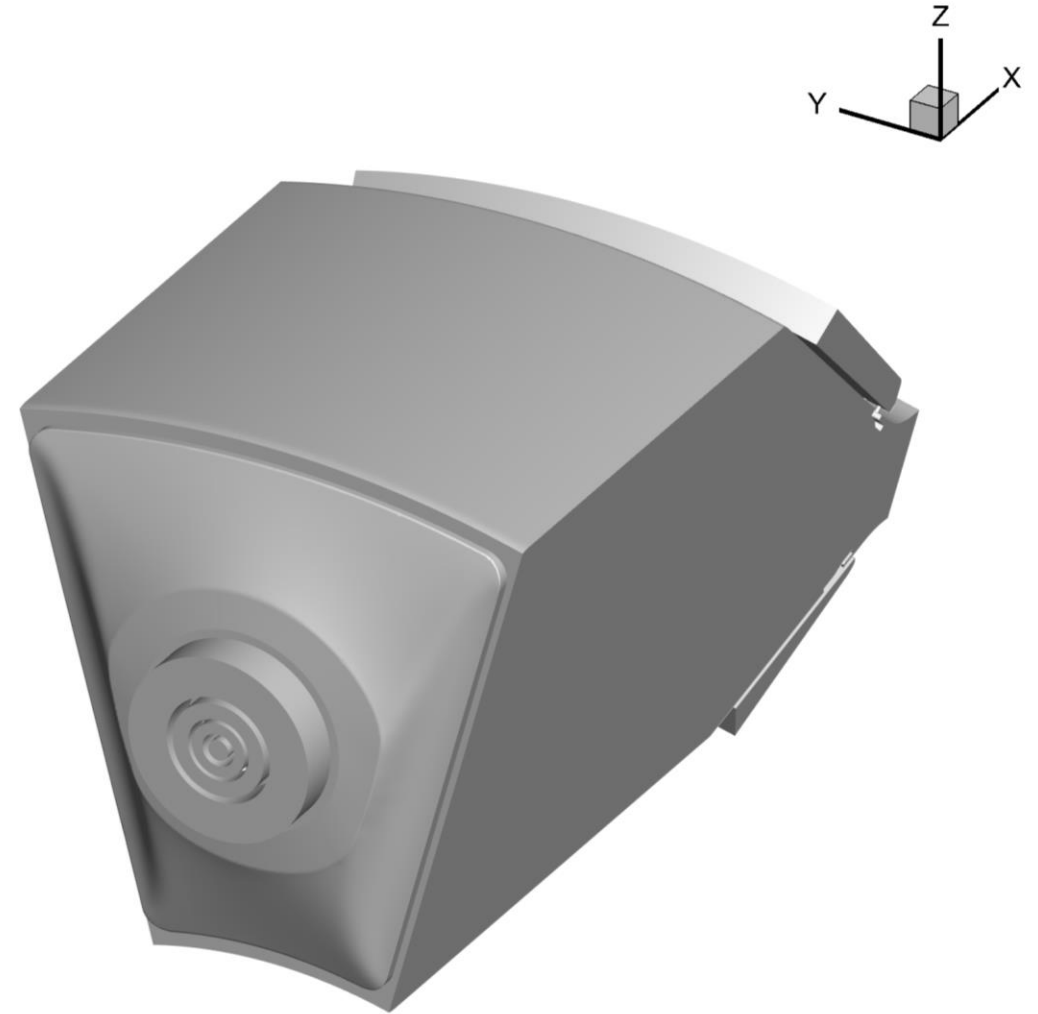
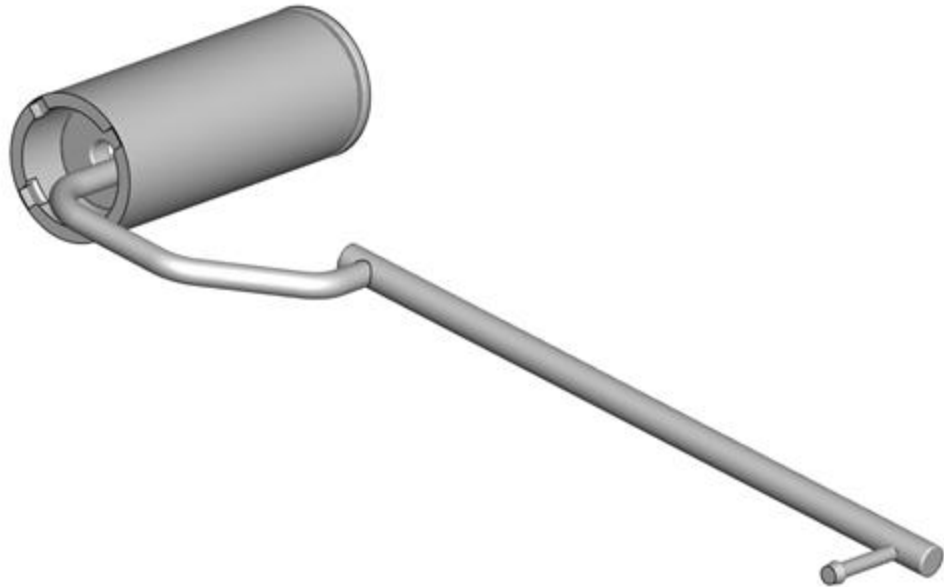
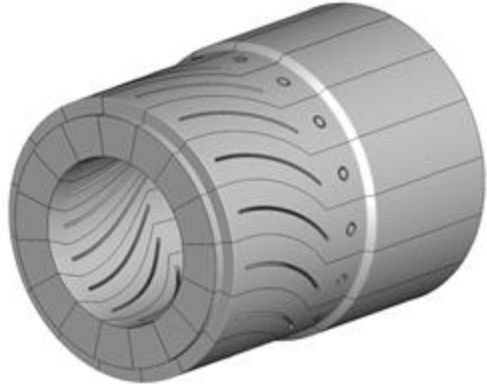


Source: [Eurostat, "Energy statistics - an overview," 2024](#)

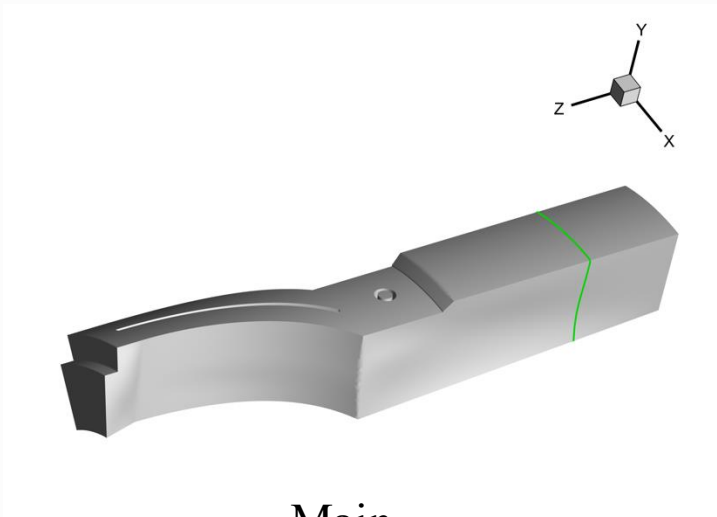
Starting Geometry



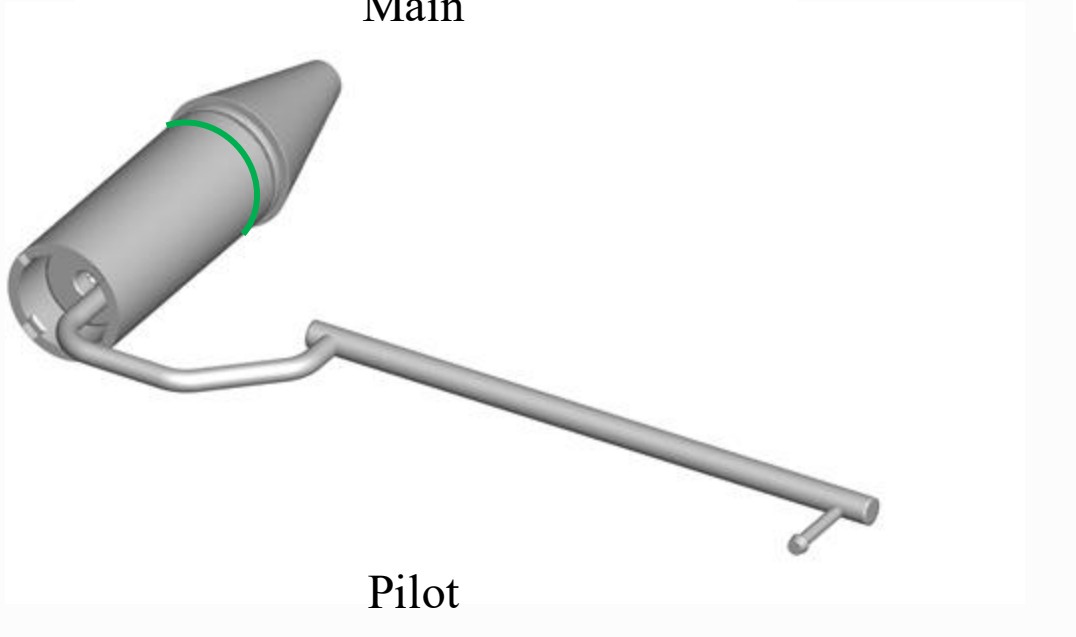
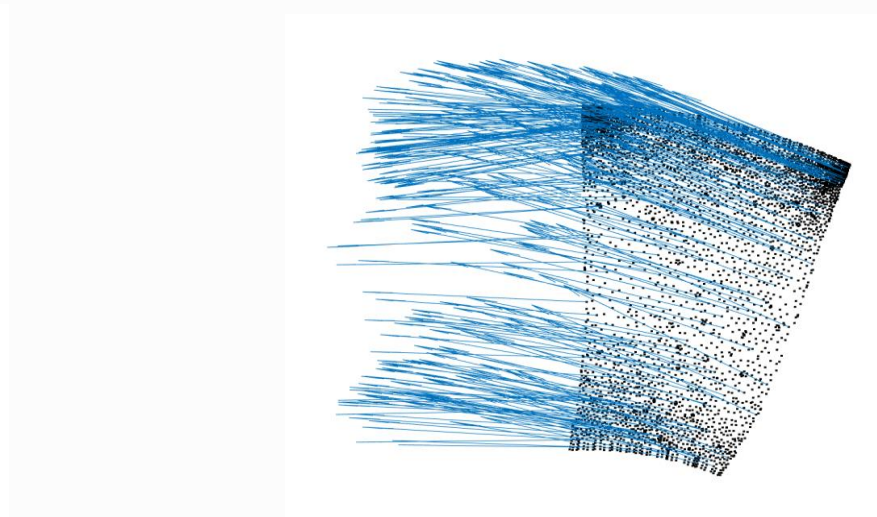
Simulated Geometry



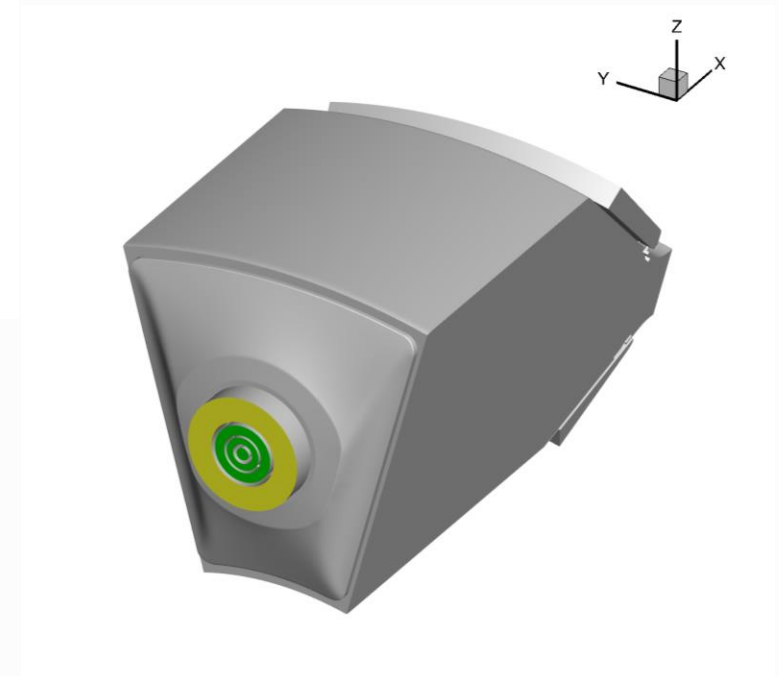
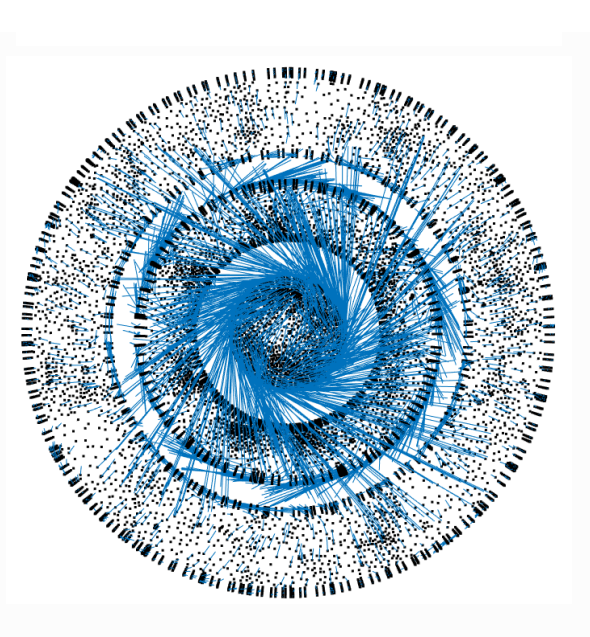
Results Reconstruction



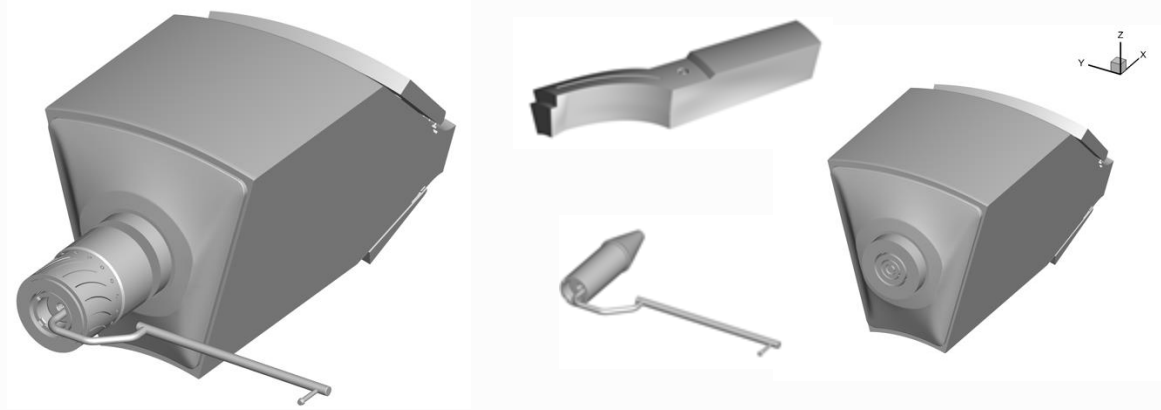
Main



Pilot



Coupling Objective



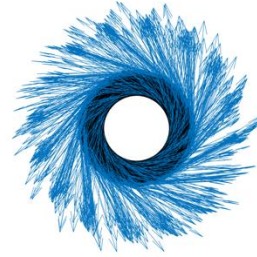
Fluid Volume	Uncoupled	Coupled
Combustion Chamber	26 million	26 million
MAIN	33 million	2 million
PILOT	17 million	17 million
Total	76 million	45 million

Achieved a 70% reduction in the number of elements

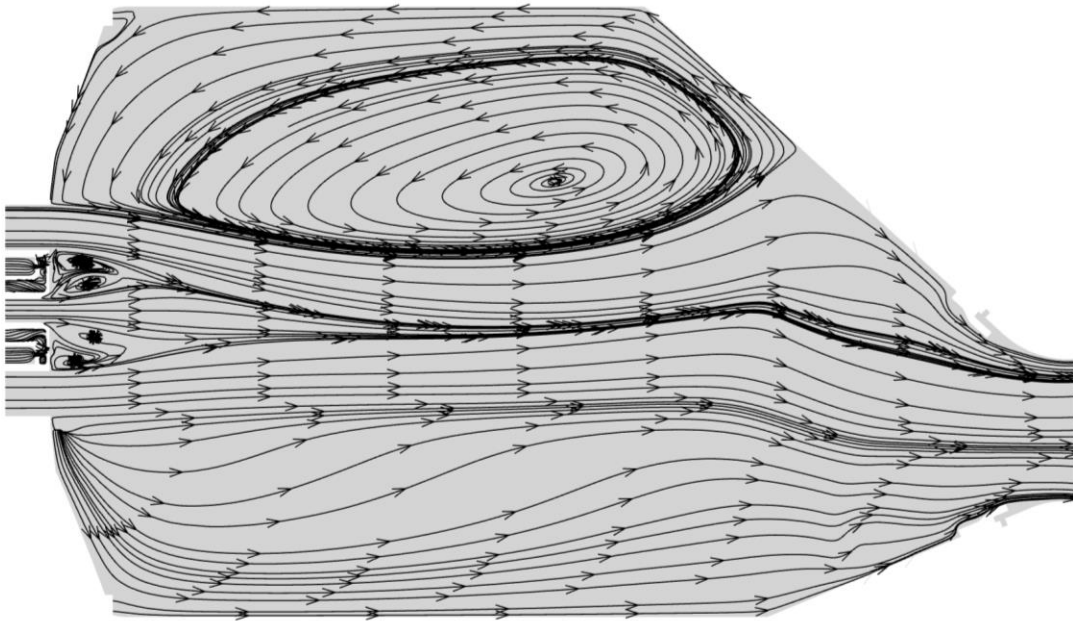
Stream Traces

Swirl Number

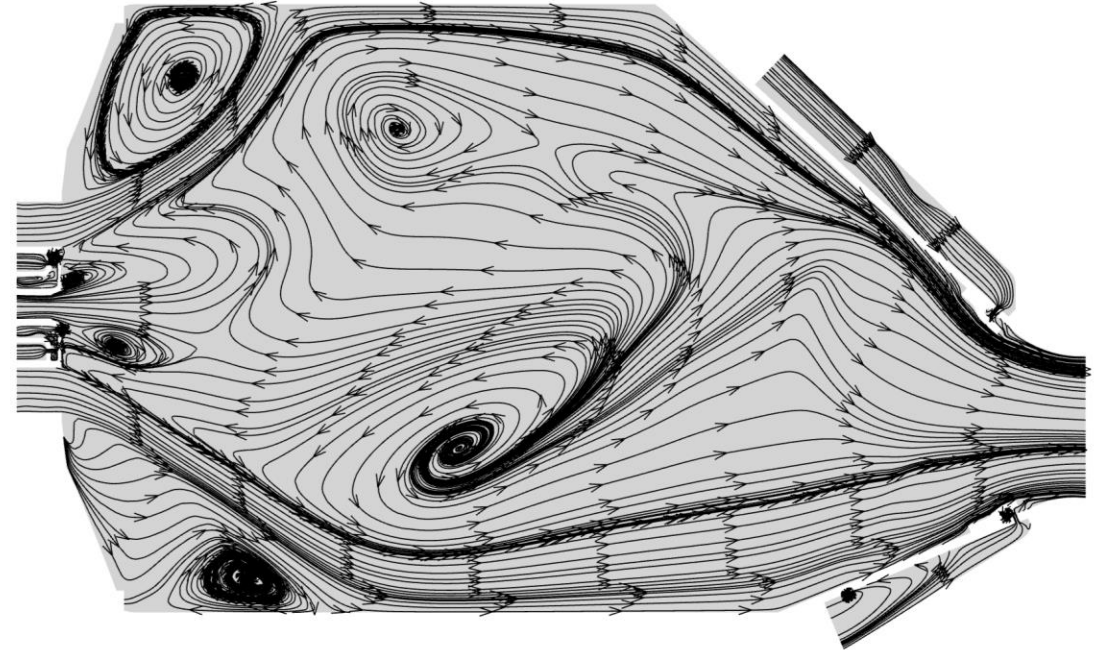
$$S = \frac{\int_{R_i}^{R_o} r \rho u_{\text{axial}} u_{\theta} dA}{R_o \int_{R_i}^{R_o} \rho u_{\text{axial}}^2 dA}$$



χ_{H_2}	0	0.25	0.35	0.5
S	30.01	29.54	29.46	29.24

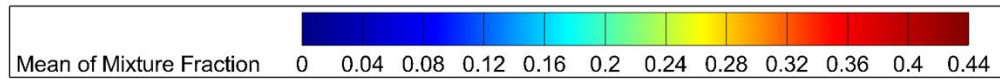
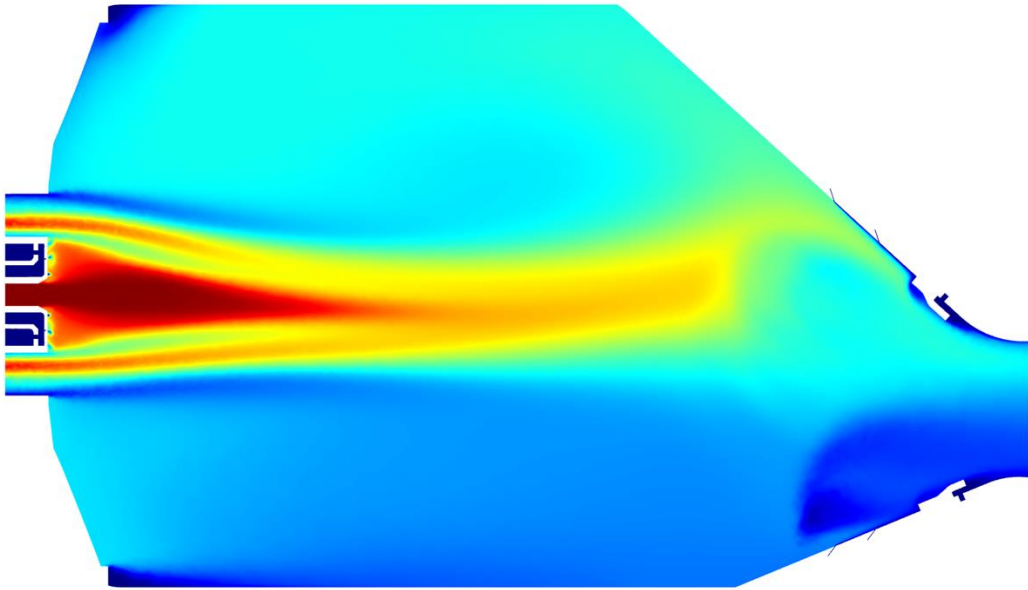


Without Swirl

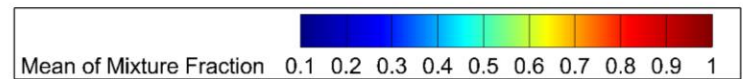
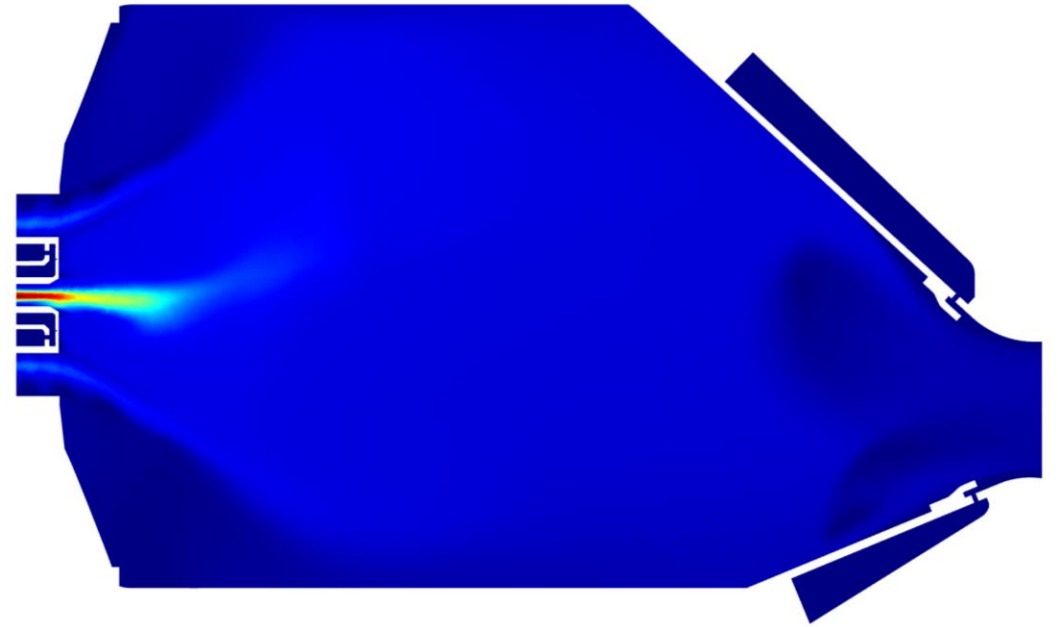


With Swirl

Fuel Distribution

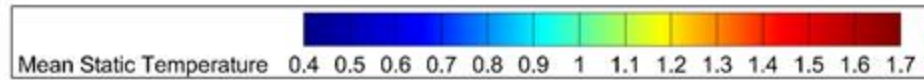
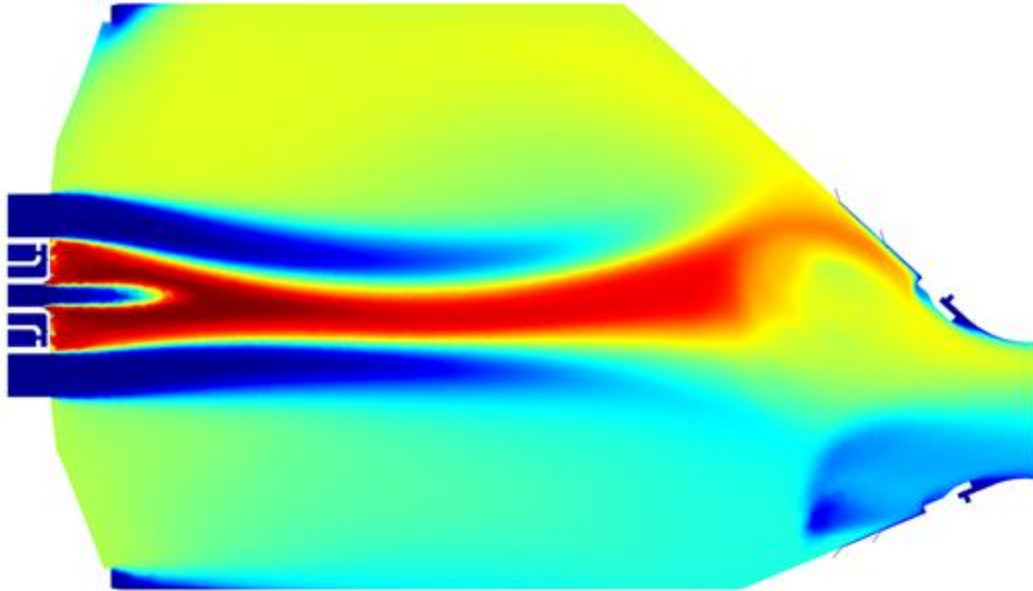


Without Swirl

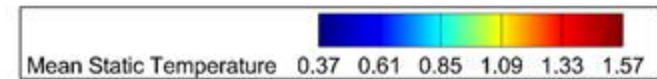
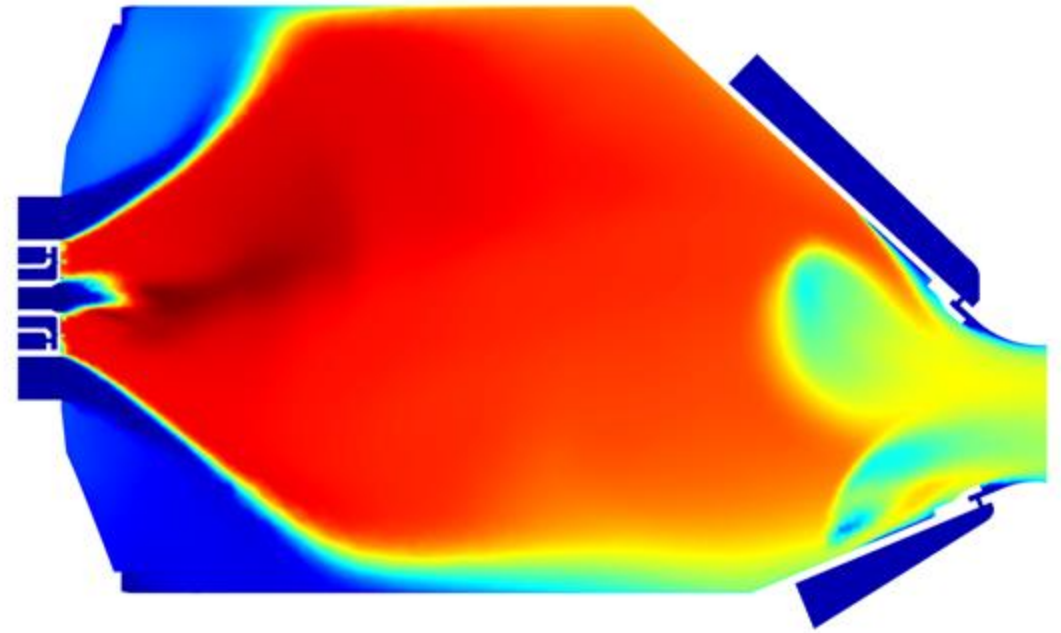


With Swirl

Temperature Distribution

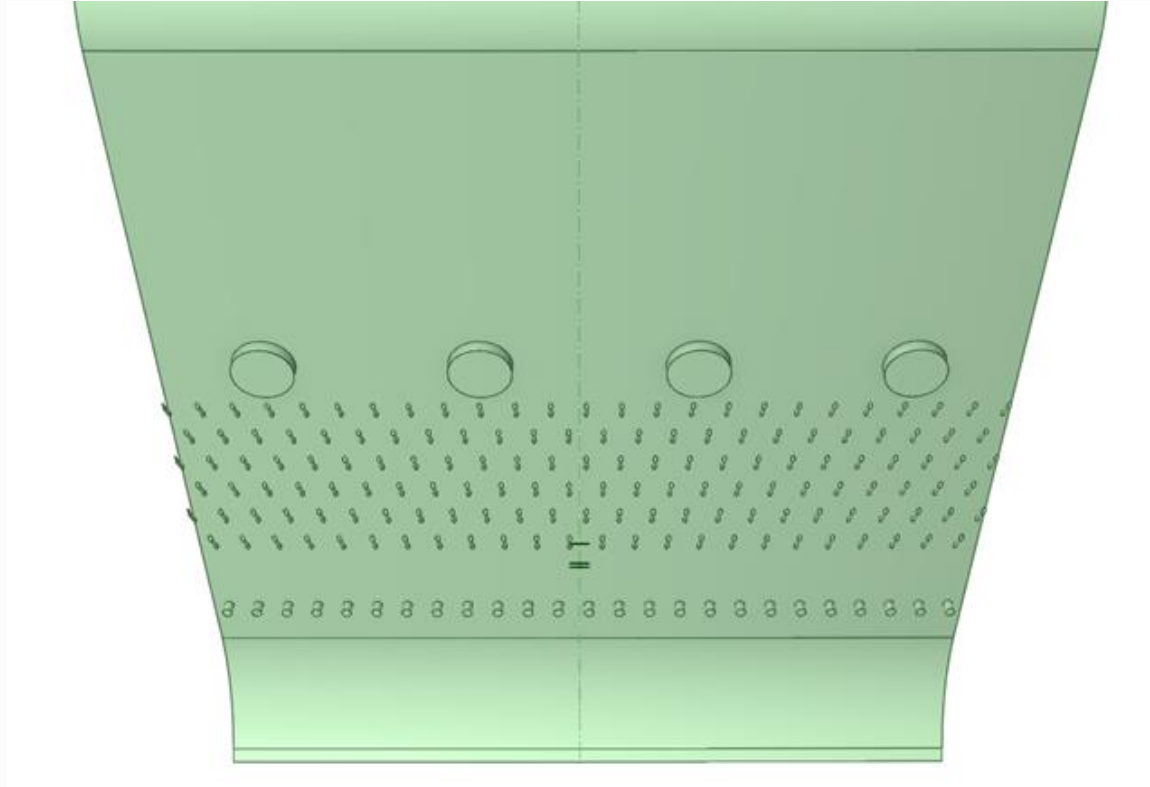


Without Swirl

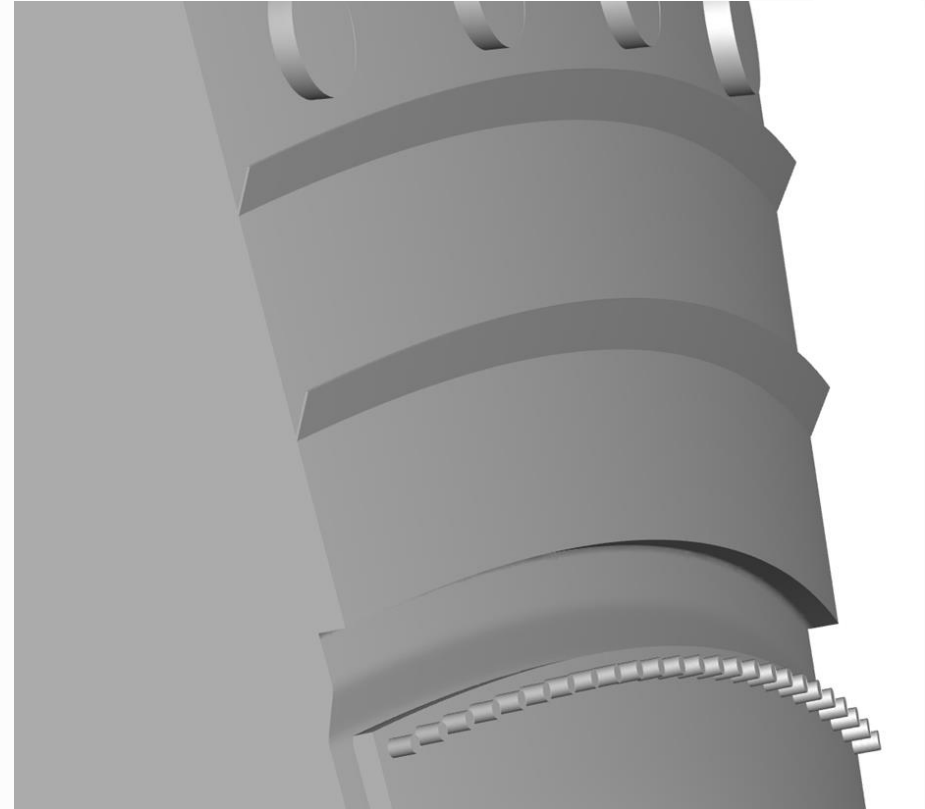


With Swirl

Effusion Holes

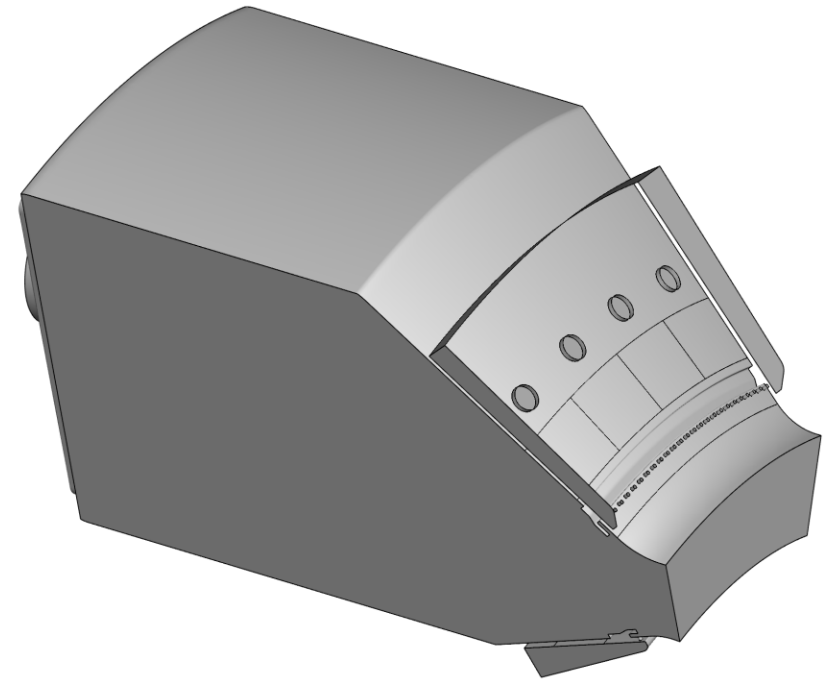
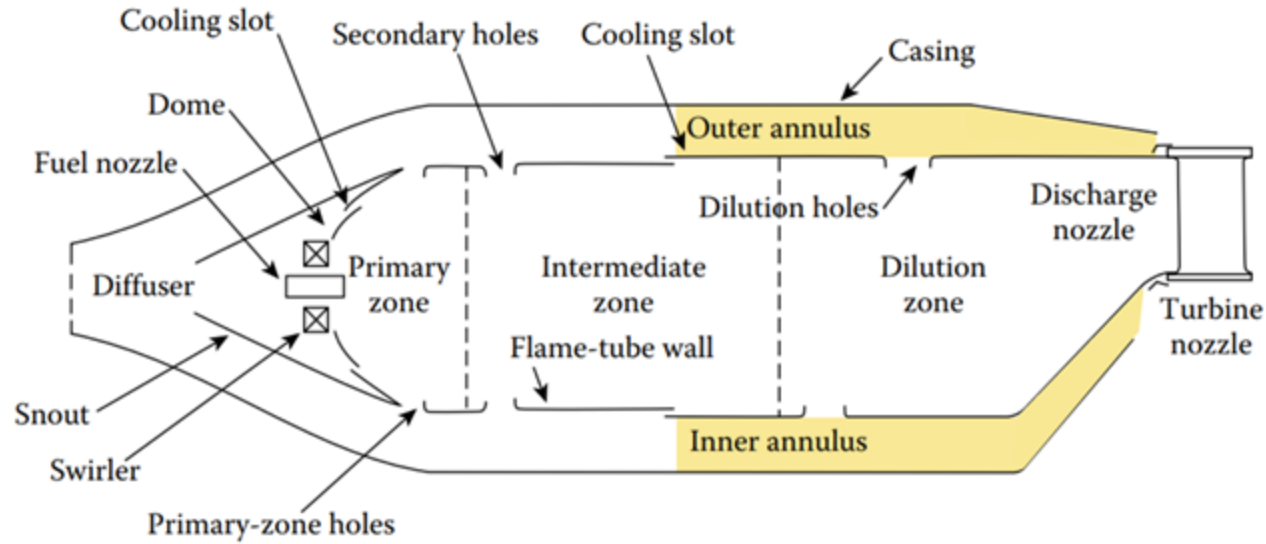


Real Geometry

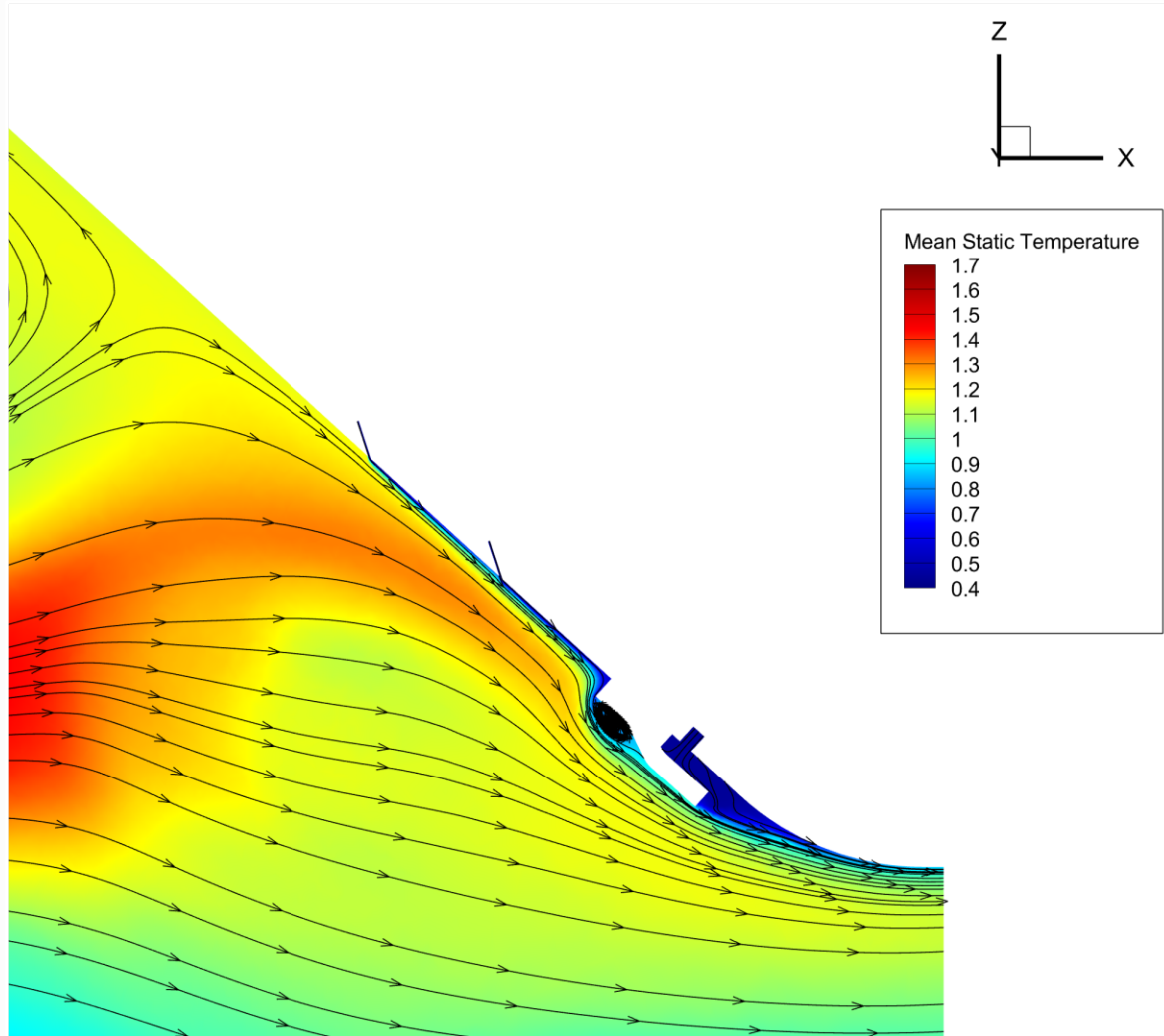


Two Cooling Stripes

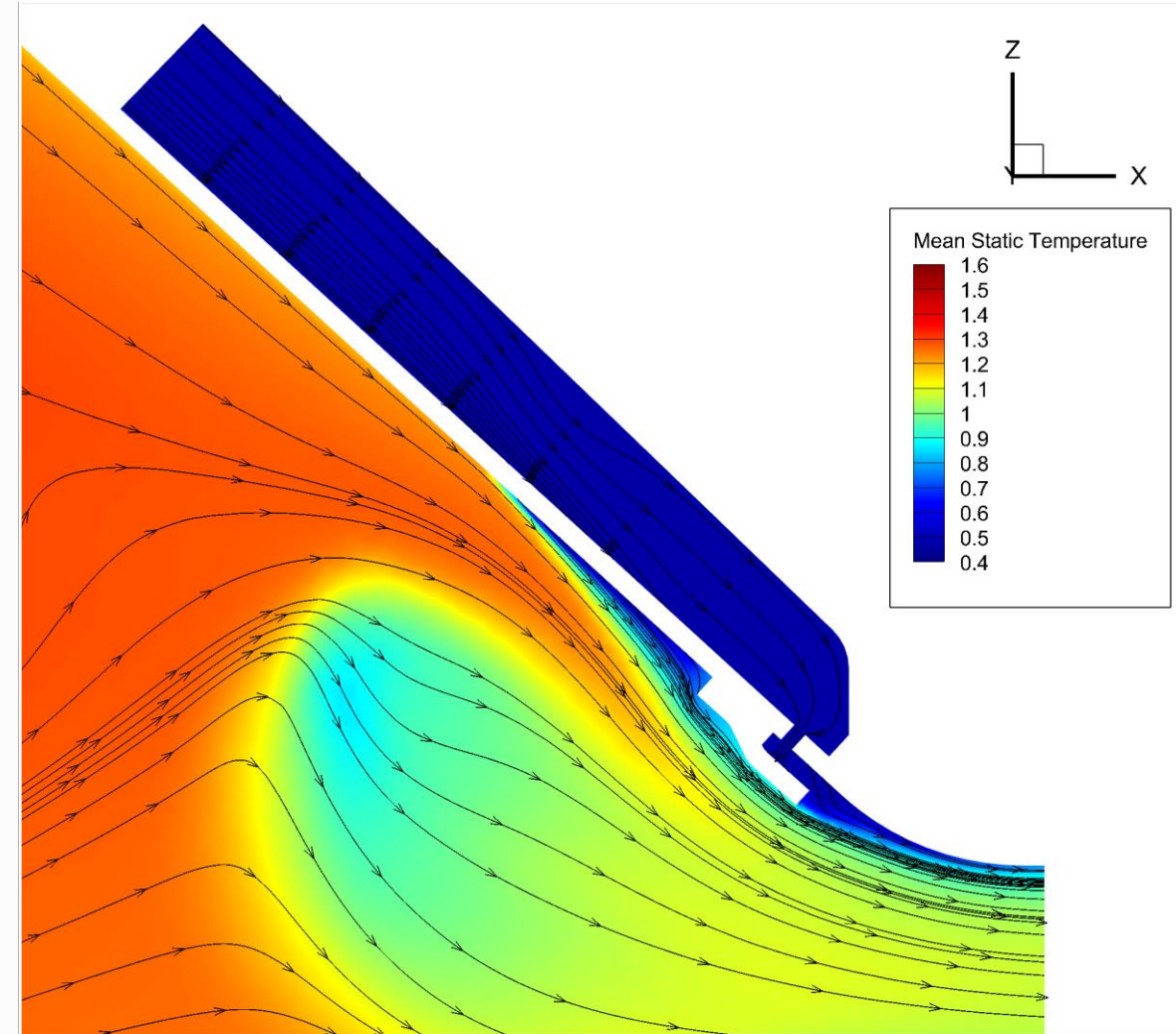
Perforated Wall Model



Wall Cooling – Film Cooling

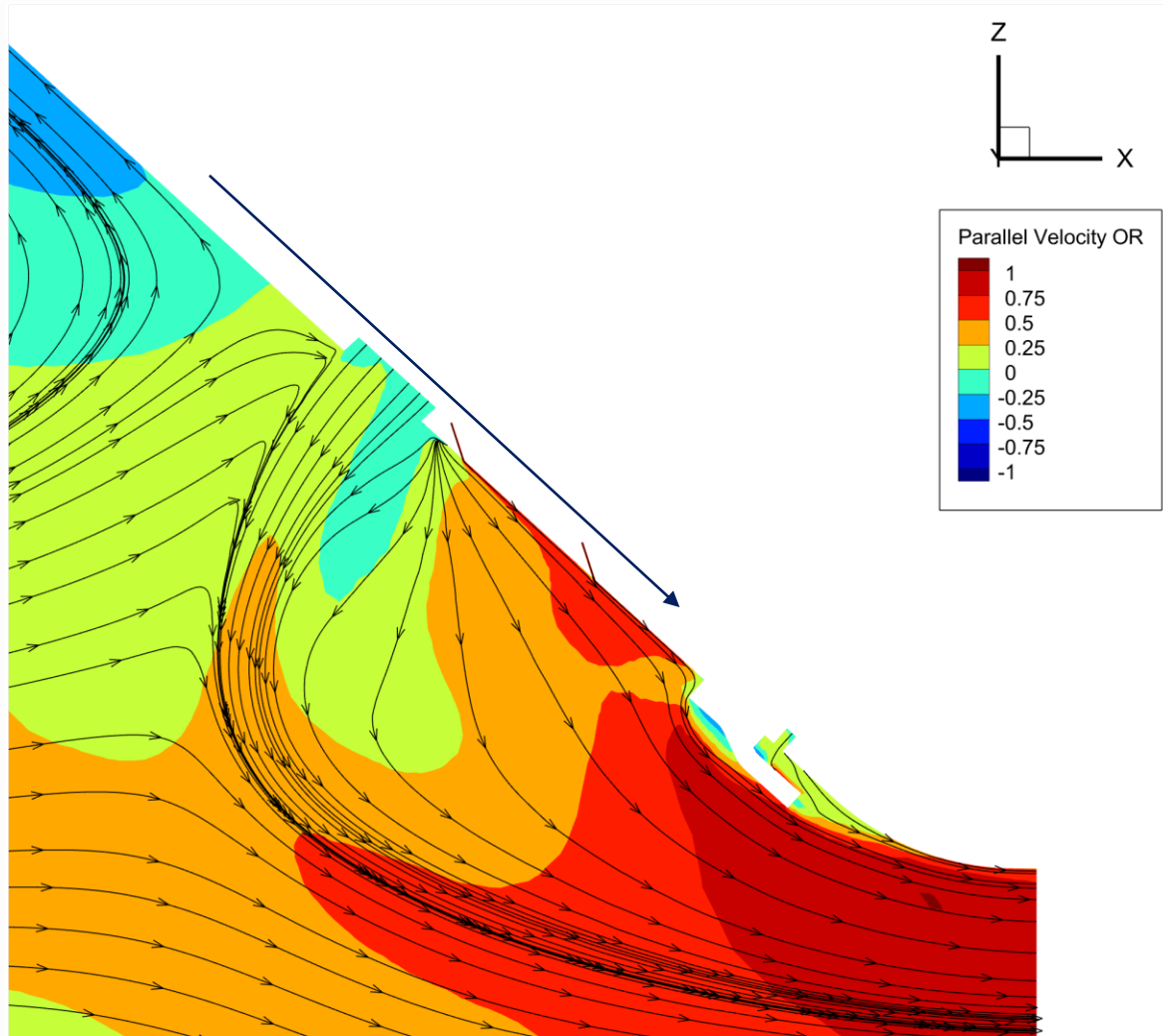


Two Cooling Stripes

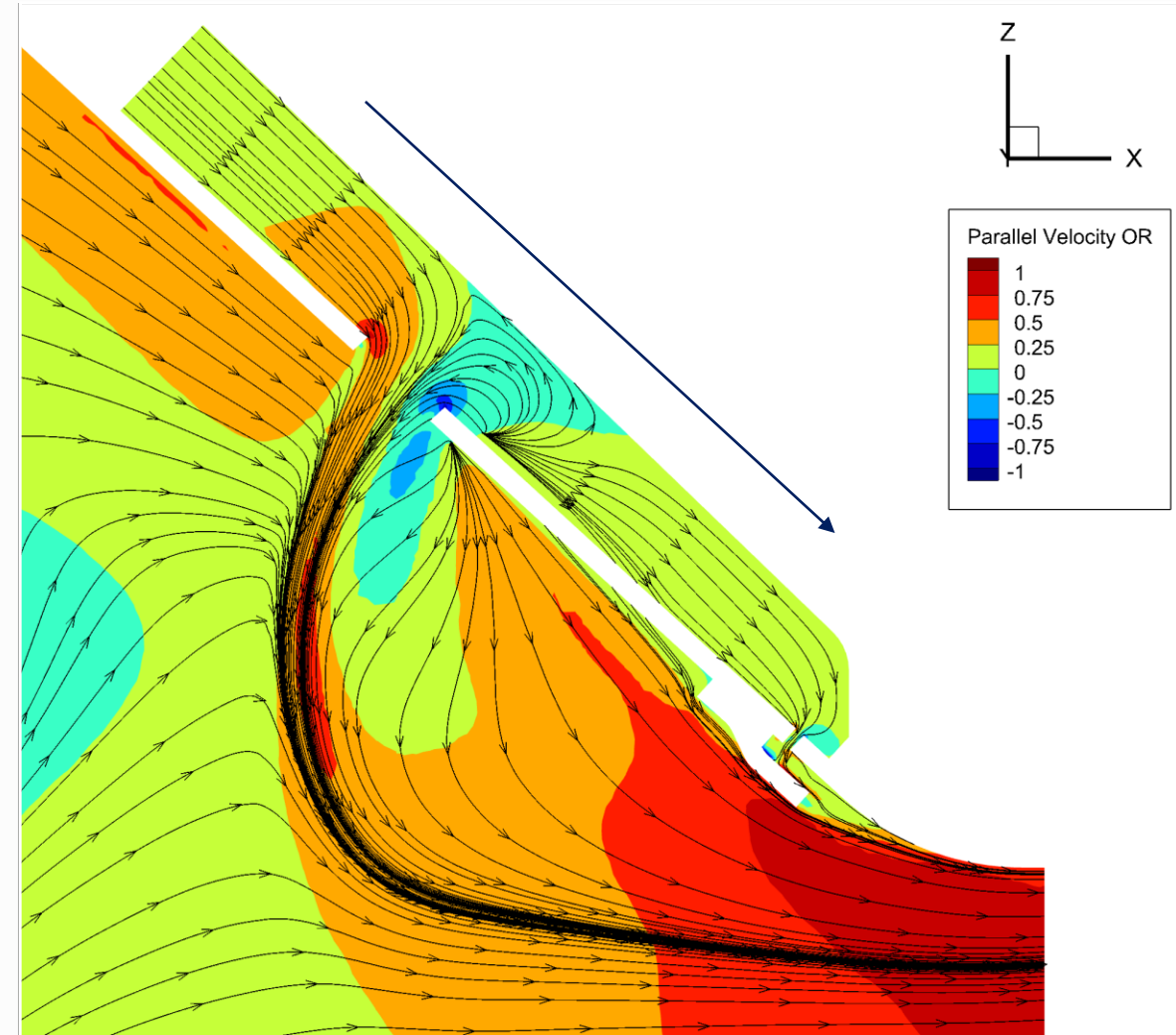


Uniform Perforated Wall Model

Dilution Holes

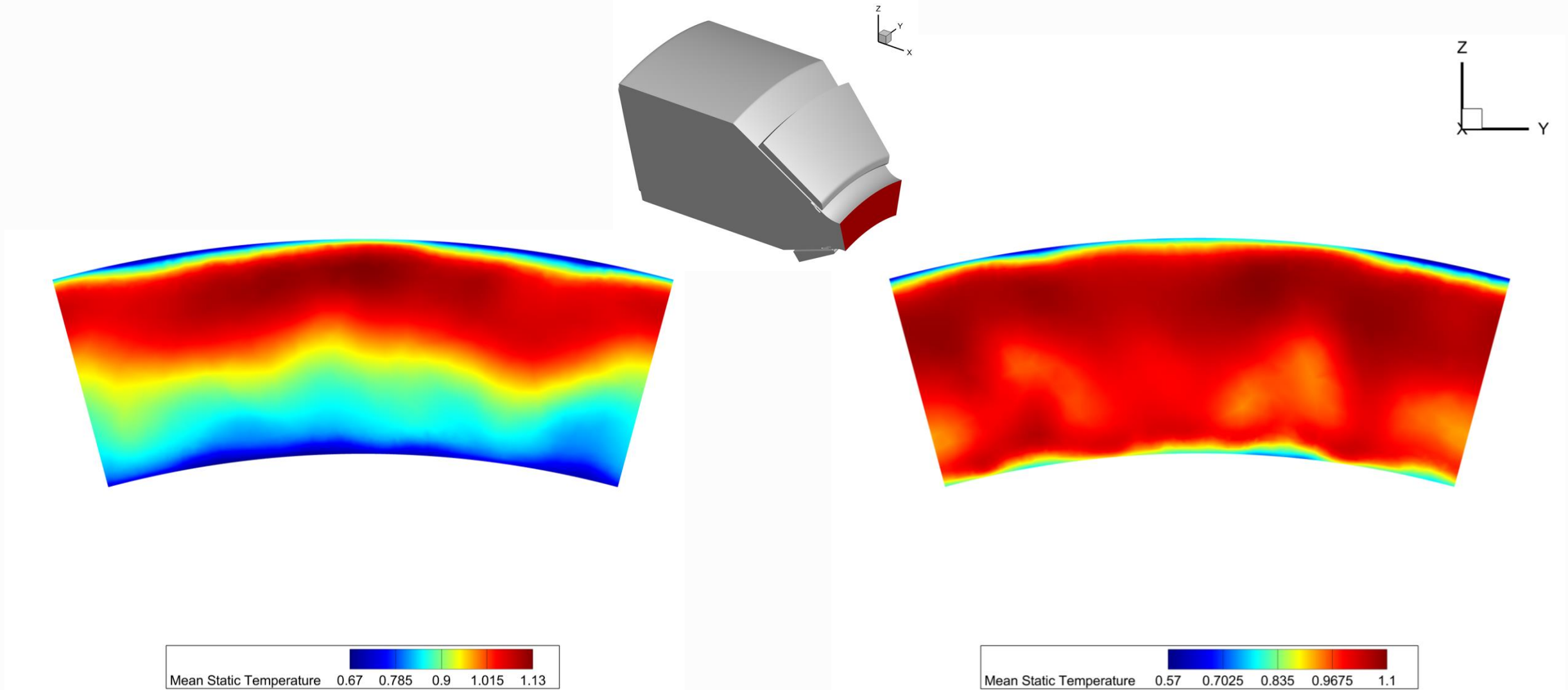


Two Cooling Stripes



Uniform Perforated Wall Model

Outlet Temperature Distribution



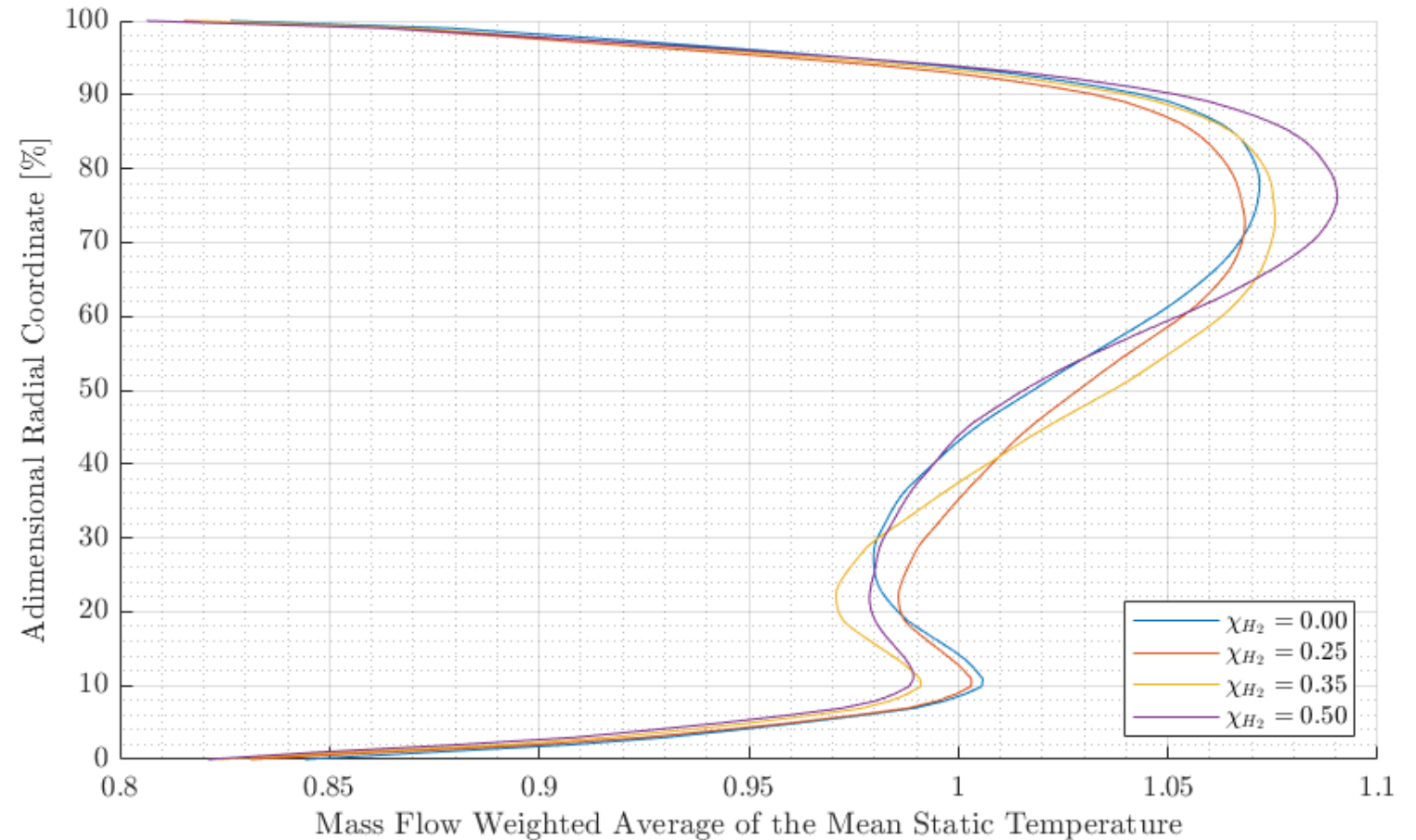
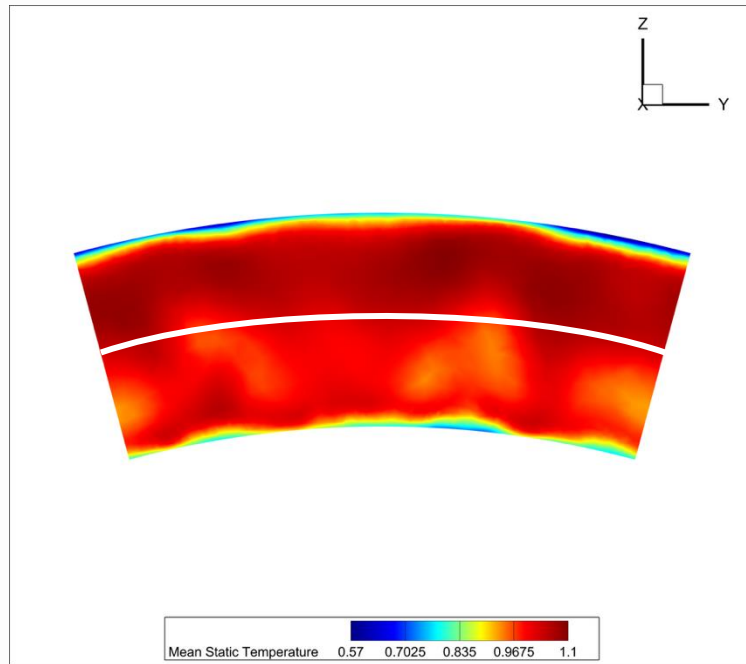
Without Swirl and with Cooling Stripes

With Swirl and with Perforated Wall Model

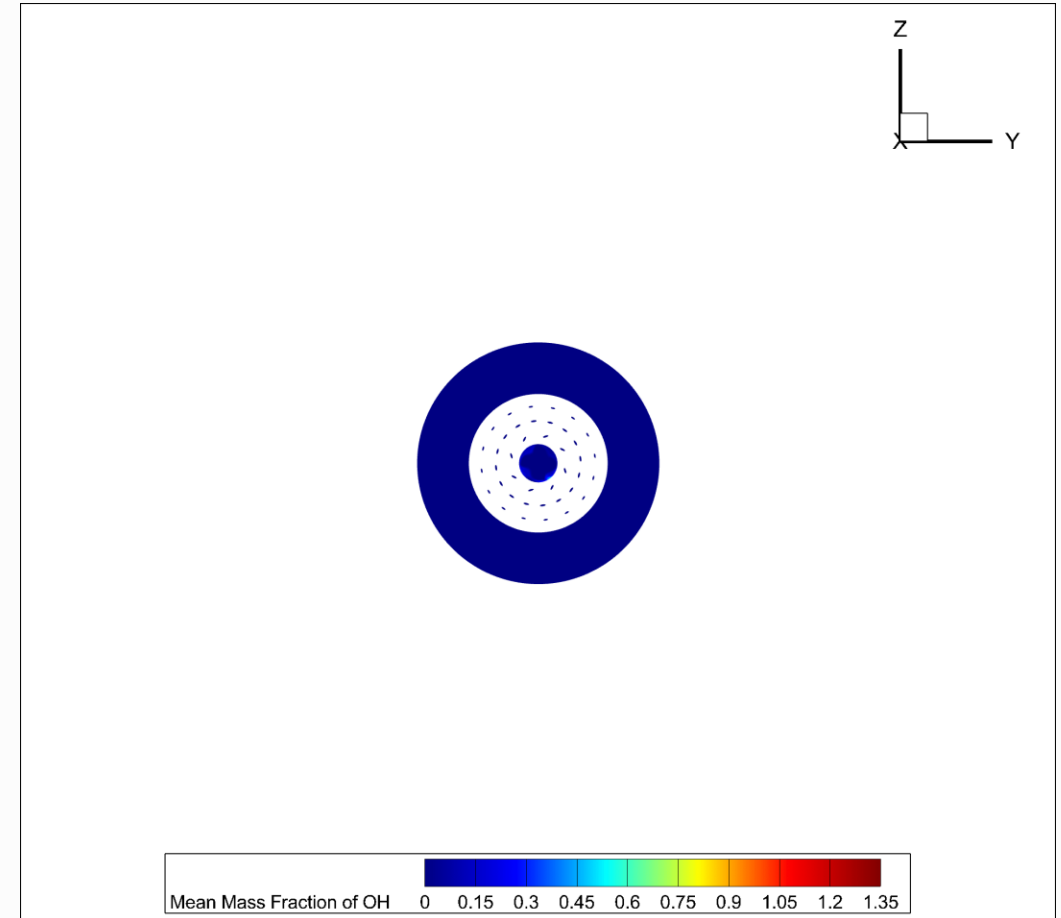
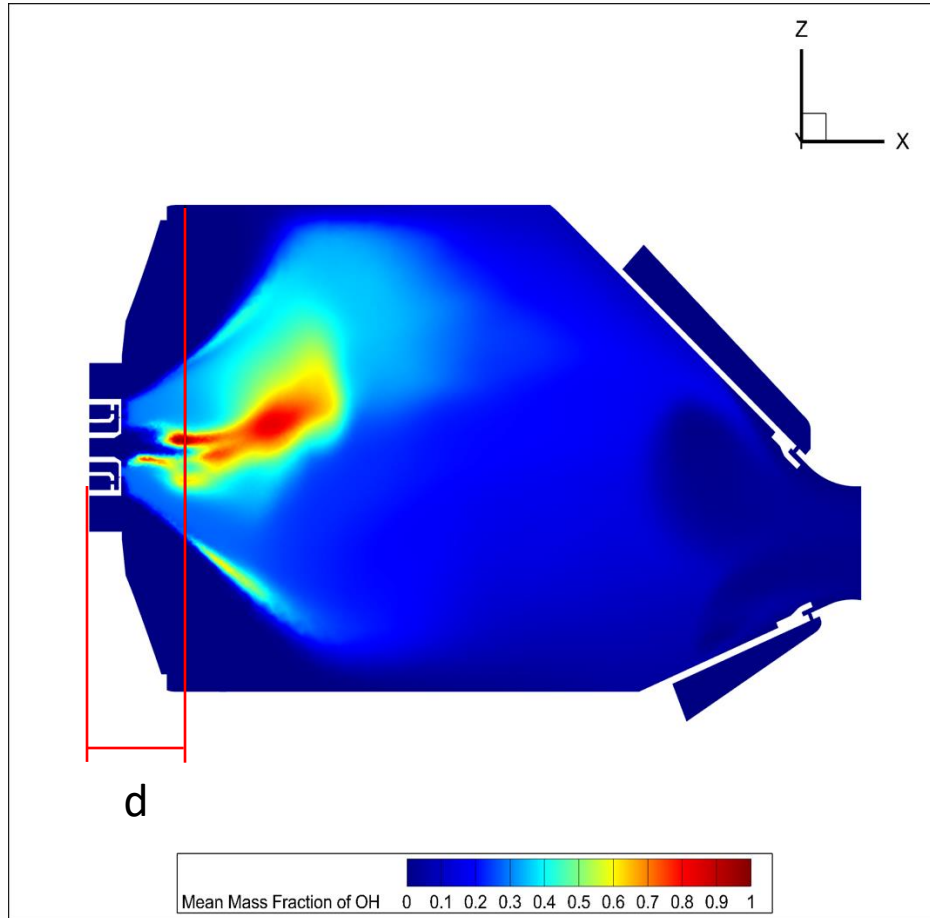
Outlet Temperature Distribution

$$PF = \frac{T_{max} - T_{out_{mean}}}{T_{out_{mean}} - T_{in}}$$

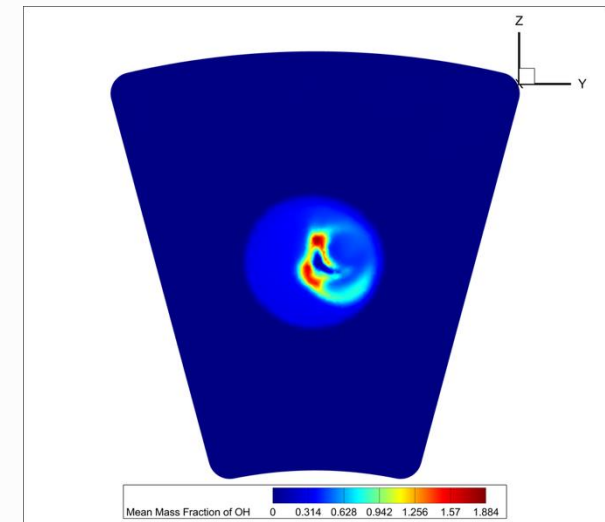
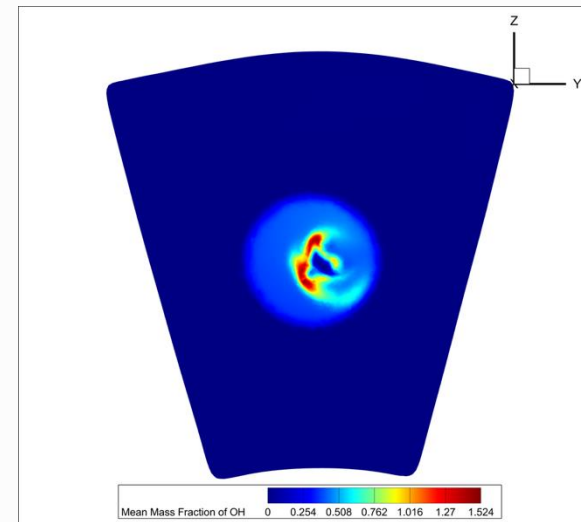
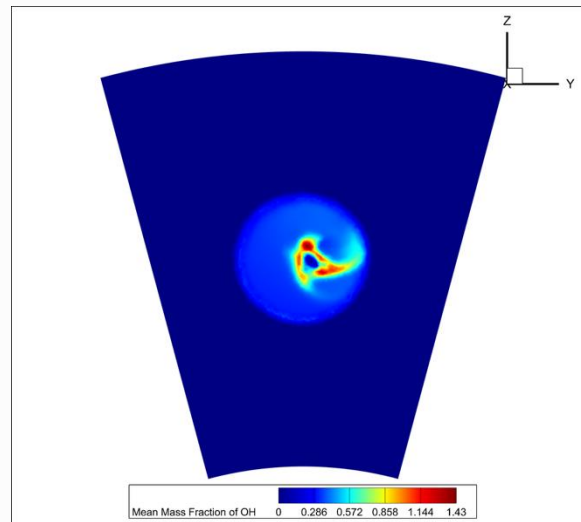
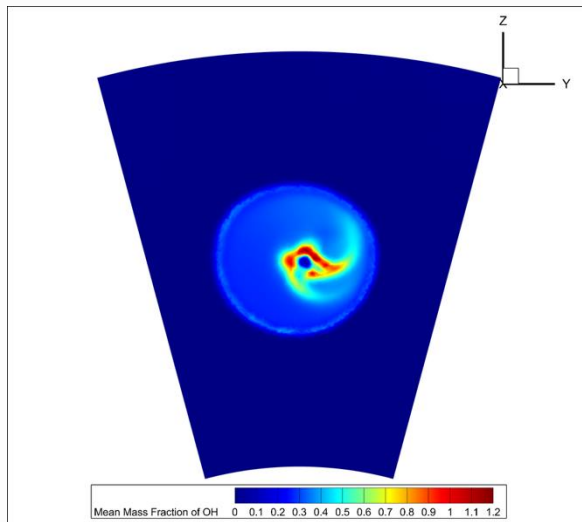
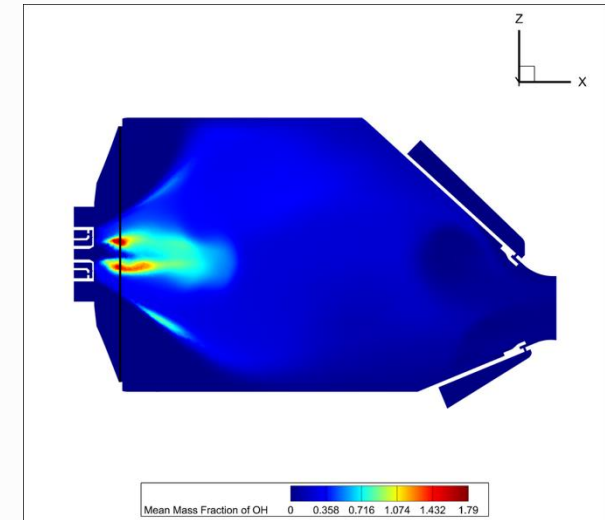
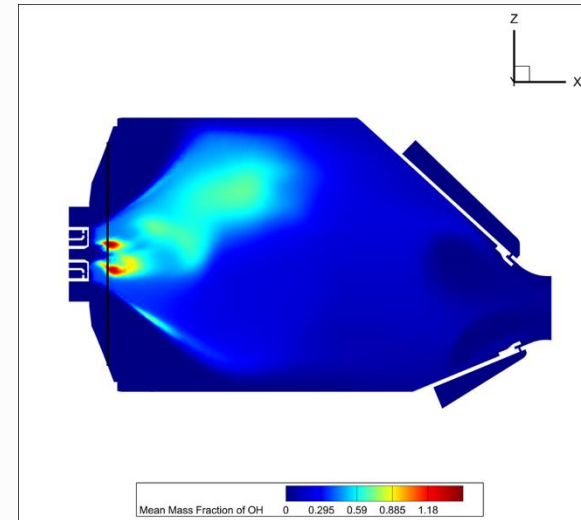
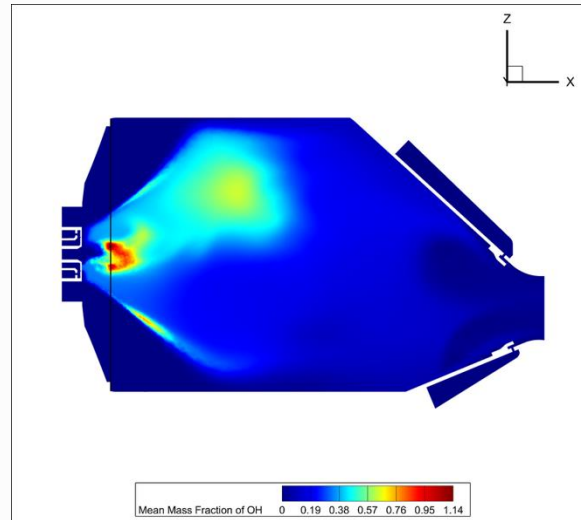
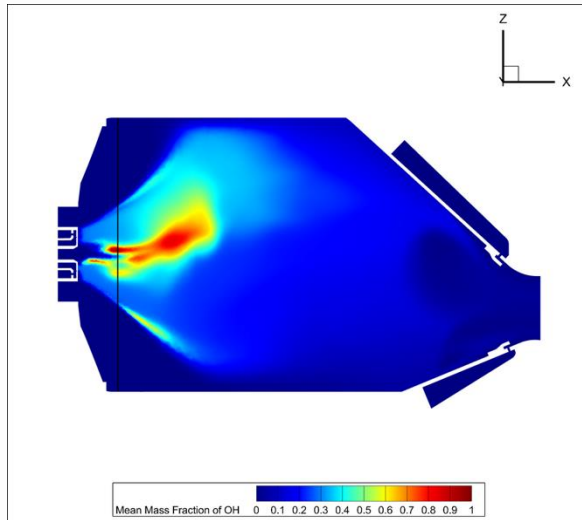
χ_{H_2}	0 %	25 %	35 %	50 %
Pattern Factor	0.1639	0.1961	0.2060	0.2339



Flame



Flame Behavior



0% H₂

25% H₂

35% H₂

50% H₂

Achievements



70% reduction in
the number of
elements



Achieved an
accurate,
functioning
combustion model



Obtained results
with four different
fuel blends

CFD Analysis of a Lean Premixed Gas Turbine Combustor for H2 Applications



**Politecnico
di Torino**

Department of Energy
"G.Ferraris"



EthosEnergy